

**MECÂNICA COMPUTACIONAL**  
Licenciatura em Engenharia Biomédica

2ª Época

Ano Lectivo de 2005/2006

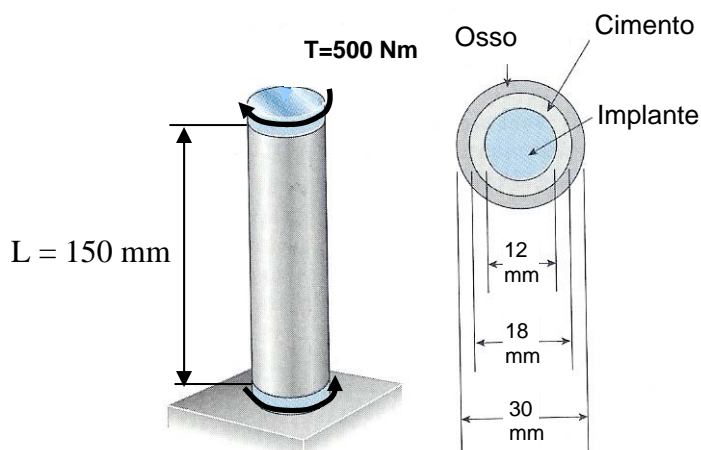
21/Julho/2006

- O Exame é sem consulta. O formulário está anexo a este exame.
- Não são permitidos computadores pessoais ou telemóveis.
- Todas as folhas do exame deverão ser identificadas.
- A duração do exame é de 2 horas e 30 minutos.

**Problema I (5 val.)**

A figura representa um modelo simplificado para analisar um implante cimentado num osso longo. Sujeitou-se o conjunto a um momento torsor  $T=500$  Nm aplicada como mostra a figura. Considere o módulo de rigidez transversal para o osso de 10 GPa, para o cimento de 1,5 GPa e para o Implante de 50 GPa.

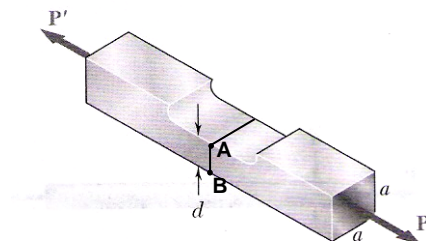
- a) Determine as tensões de corte máximas no implante, no cimento e no osso compacto.
- b) Considere todos os materiais com rigidez ao corte igual à do osso (10 GPa). Nestas condições determine a tensão de corte máxima.
- c) Determine o ângulo de torção para a situação da alínea a) e para a situação da alínea b)



**Problema II (4 val.)**

Retirou-se um porção de material a uma barra de secção transversal quadrada com  $a=30$  mm, obtendo-se o componente representado na figura. O carregamento  $P$  está alinhado com o eixo da peça original.

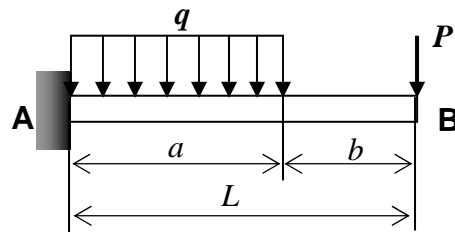
- a) Para  $P = 5$  KN e  $d = 20$  mm determine a tensão normal no pontos A e B.
- b) Para  $P=15$  KN e tensão normal admissível de 55 MPa, determine o valor mínimo da espessura  $d$ .



### Problema III (6 val.)

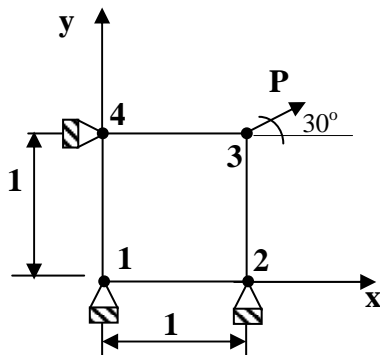
Considere o varão de alumínio (secção circular) cujo módulo de rigidez é  $E=70$  GPa, sujeito ao carregamento representado na figura. Considere  $L=2$  metros,  $a=1.4$  e  $b=0.6$ . O carregamento distribuído é de  $q=1000$  N/m e a carga concentrada  $P=1500$  N.

- Calcule as reacções no apoio.
- Desenhe os diagramas de esforço transversal e momento flector, e indique os respectivos valores máximos absolutos.
- Calcule a equação da curva elástica da viga (em função de  $EI$ ).
- Sabendo que o deslocamento vertical em B não pode exceder 20 mm, determine o diâmetro mínimo que o varão pode ter.



### Problema IV (5 val.)

Considere o elemento plano de quatro nós, apresentado na figura, para um problema de deformação plana. Admita  $E=1$  e  $\nu=0$ .



- Estabeleça as funções de base para este elemento (no referencial  $x-y$ ).
- Admita que os elementos da diagonal da matriz de rigidez valem  $1/2$  e os elementos fora da diagonal  $1/8$ , determine os deslocamentos nodais para as condições indicadas na figura.
- Determine a tensão no centro do elemento.

## FORMULÁRIO:

$$\sigma = \frac{P}{A} \quad \sigma = \lim_{\Delta A \rightarrow 0} \frac{\Delta F}{\Delta A} \quad \tau = \lim_{\Delta A \rightarrow 0} \frac{\Delta V}{\Delta A} \quad \tau_{xy} = \tau_{yx} \quad \sigma = \frac{P}{A_0} \cos^2 \theta$$

$$\tau = \frac{P}{A_0} \cos \theta \sin \theta \quad \varepsilon = \frac{d\delta}{dx} \quad \sigma = E\varepsilon \quad \varepsilon_T = \alpha \Delta T \quad \delta = \frac{PL}{AE} \quad \delta = \sum_i \frac{P_i L_i}{A_i E_i}$$

$$\delta = \int_0^L \frac{P}{AE} dx \quad \varepsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E} \quad \varepsilon_y = -\nu \frac{\sigma_x}{E} + \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E} \quad \varepsilon_z = -\nu \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} + \frac{\sigma_z}{E}$$

$$\tau_{xy} = G\gamma_{xy} \quad \tau_{yz} = G\gamma_{yz} \quad \tau_{xz} = G\gamma_{xz} \quad G = \frac{E}{2(1+\nu)} \quad \gamma = \frac{\rho\phi}{L} \quad \tau = \frac{T\rho}{J}$$

$$\phi = \frac{TL}{JG} \quad \phi = \sum_i \frac{T_i L_i}{J_i G_i} \quad \phi = \int_0^L \frac{T}{JG} dx \quad \frac{1}{\rho} = \frac{M}{EI} \quad \frac{1}{\rho'} = \frac{\nu}{\rho} \quad \sigma_x = -\frac{My}{I}$$

$$\sigma_x = \frac{P}{A} - \frac{My}{I} \quad \sigma_x = \frac{P}{A} - \frac{M_z y}{I_z} + \frac{M_y z}{I_y} \quad \tan \phi = \frac{I_z}{I_y} \tan \theta \quad \frac{dV}{dx} = -w$$

$$V_D - V_c = -\int_{x_c}^{x_D} w dx \quad \frac{dM}{dx} = V \quad M_D - M_c = \int_{x_c}^{x_D} V dx \quad \frac{1}{\rho} = \frac{M(x)}{EI} \quad \frac{d^2 y}{dx^2} = \frac{M(x)}{EI}$$

$$K_{ij}^e = \int_0^{h^e} EA \frac{d\phi_i}{dx} \frac{d\phi_j}{dx} dx \quad F_i^e = \int_0^{h^e} f \phi_i dx \quad [K_{ij}^e] = \frac{E^e A^e}{L^e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad \{F_i^e\} = \frac{f^e L^e}{2} \begin{Bmatrix} 1 \\ 1 \end{Bmatrix}$$

$$\phi_1 = 1 - \frac{x}{h^e} \quad \phi_2 = \frac{x}{h^e} \quad K_{ij}^e = \int_0^{h^e} EI \frac{d^2 \phi_i}{dx^2} \frac{d^2 \phi_j}{dx^2} dx \quad F_i^e = \int_0^{h^e} q \phi_i dx$$

$$\phi_1 = 1 - 3\left(\frac{x}{h^e}\right)^2 + 2\left(\frac{x}{h^e}\right)^3 \quad \phi_2 = -x\left(1 - \frac{x}{h^e}\right)^2 \quad \phi_3 = 3\left(\frac{x}{h^e}\right)^2 - 2\left(\frac{x}{h^e}\right)^3 \quad \phi_4 = -x\left[\left(\frac{x}{h^e}\right)^2 - \frac{x}{h^e}\right]$$

$$[K_{ij}^e] = \frac{2EI}{h_e^3} \begin{bmatrix} 6 & -3h_e & -6 & -3h_e \\ -3h_e & 2h_e^2 & 3h_e & h_e^2 \\ -6 & 3h_e & 6 & 3h_e \\ -3h_e & h_e^2 & 3h_e & 2h_e^2 \end{bmatrix} \quad \{F_i^e\} = \frac{qh_e}{12} \begin{Bmatrix} 6 \\ -h_e \\ 6 \\ h_e \end{Bmatrix} \quad K_{\alpha\beta}^e = \int_{\Omega^e} k_{ij} \frac{\partial \phi_\alpha}{\partial x_i} \frac{\partial \phi_\beta}{\partial x_j} d\Omega \quad F_\alpha^e = \int_{\Gamma_e} q \phi_\alpha d\Gamma$$

$$\phi_1 = \frac{1}{4}(1-\xi)(1-\eta) \quad \phi_2 = \frac{1}{4}(1+\xi)(1-\eta) \quad \phi_3 = \frac{1}{4}(1+\xi)(1+\eta) \quad \phi_4 = \frac{1}{4}(1-\xi)(1+\eta)$$


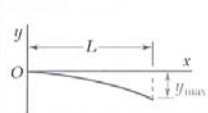
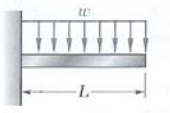
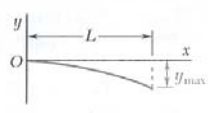


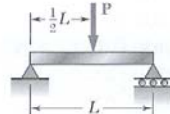
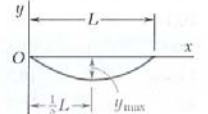
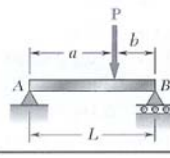
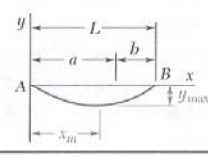
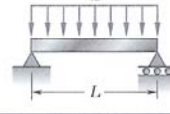
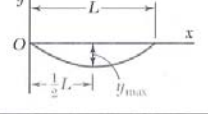
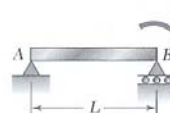
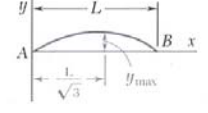
deformação plana

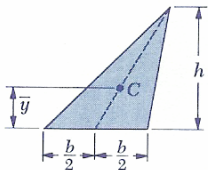
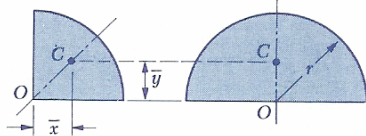
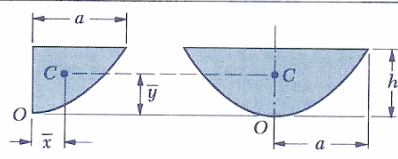
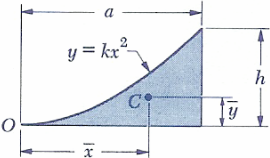
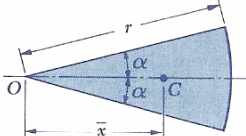
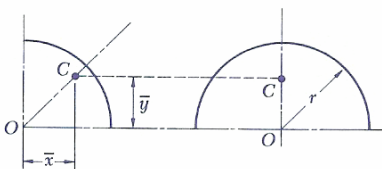
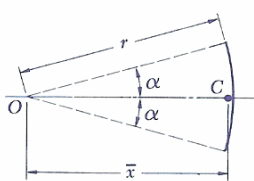
$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix} \begin{Bmatrix} \varepsilon_x \\ \varepsilon_y \\ \gamma_{xy} \end{Bmatrix}$$

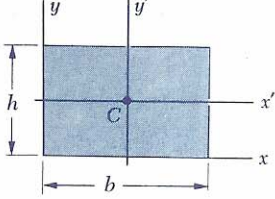
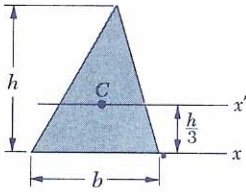
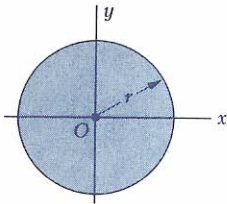
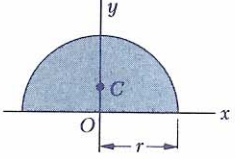
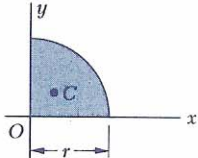
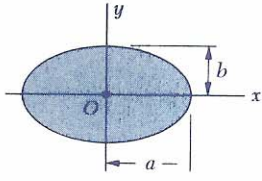
tensão plana

$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} = \frac{E}{(1-\nu^2)} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{Bmatrix} \varepsilon_x \\ \varepsilon_y \\ \gamma_{xy} \end{Bmatrix}$$

$$\varepsilon_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \quad \varepsilon_{ij} = \frac{1}{2} \gamma_{ij}$$

Viga e carregamento	Linha elástica	Flèche máxima	Rotação na extremidade	Equação da linha elástica
		$-\frac{PL^3}{3EI}$	$-\frac{PL^2}{2EI}$	$y = \frac{P}{6EI}(x^3 - 3Lx^2)$
		$-\frac{pL^4}{8EI}$	$-\frac{pL^3}{6EI}$	$y = -\frac{P}{24EI}(x^4 - 4Lx^3 + 6L^2x^2)$
		$-\frac{ML^2}{2EI}$	$-\frac{ML}{EI}$	$y = -\frac{M}{2EI}x^2$
		$-\frac{PL^3}{48EI}$	$\pm \frac{PL^2}{16EI}$	Para $x \leq \frac{1}{2}L$ : $y = \frac{P}{48EI}(4x^3 - 3L^2x)$
		Para $a > b$ : $-\frac{Pb(L^2 - b^2)^{3/2}}{9\sqrt{3}EIL}$ em $x_m = \sqrt{\frac{L^2 - b^2}{3}}$	$\theta_A = -\frac{Pb(L^2 - b^2)}{6EIL}$ $\theta_B = +\frac{Pa(L^2 - a^2)}{6EIL}$	Para $x < a$ : $y = \frac{Pb}{6EIL}[x^3 - (L^2 - b^2)x]$ Para $x = a$ : $y = -\frac{Pa^2b^2}{3EIL}$
		$-\frac{5pL^4}{384EI}$	$\pm \frac{pL^3}{24EI}$	$y = -\frac{P}{24EI}(x^4 - 2Lx^3 + L^3x)$
		$\frac{ML^2}{9\sqrt{3}EI}$	$\theta_A = +\frac{ML}{6EI}$ $\theta_B = -\frac{ML}{3EI}$	$y = -\frac{M}{6EIL}(x^3 - L^2x)$

Shape		$\bar{x}$	$\bar{y}$	Area
Triangular area			$\frac{h}{3}$	$\frac{bh}{2}$
Quarter-circular area		$\frac{4r}{3\pi}$	$\frac{4r}{3\pi}$	$\frac{\pi r^2}{4}$
Semicircular area		0	$\frac{4r}{3\pi}$	$\frac{\pi r^2}{2}$
Semiparabolic area		$\frac{3a}{8}$	$\frac{3h}{5}$	$\frac{2ah}{3}$
Parabolic area		0	$\frac{3h}{5}$	$\frac{4ah}{3}$
Parabolic spandrel		$\frac{3a}{4}$	$\frac{3h}{10}$	$\frac{ah}{3}$
Circular sector		$\frac{2r \sin \alpha}{3\alpha}$	0	$\alpha r^2$
Quarter-circular arc		$\frac{2r}{\pi}$	$\frac{2r}{\pi}$	$\frac{\pi r}{2}$
Semicircular arc		0	$\frac{2r}{\pi}$	$\pi r$
Arc of circle		$\frac{r \sin \alpha}{\alpha}$	0	$2\alpha r$

Rectangle		$\bar{I}_{x'} = \frac{1}{12}bh^3$ $\bar{I}_{y'} = \frac{1}{12}b^3h$ $I_x = \frac{1}{3}bh^3$ $I_y = \frac{1}{3}b^3h$ $J_C = \frac{1}{12}bh(b^2 + h^2)$
Triangle		$\bar{I}_{x'} = \frac{1}{36}bh^3$ $I_x = \frac{1}{12}bh^3$
Circle		$\bar{I}_x = \bar{I}_y = \frac{1}{4}\pi r^4$ $J_O = \frac{1}{2}\pi r^4$
Semicircle		$I_x = I_y = \frac{1}{8}\pi r^4$ $J_O = \frac{1}{4}\pi r^4$
Quarter circle		$I_x = I_y = \frac{1}{16}\pi r^4$ $J_O = \frac{1}{8}\pi r^4$
Ellipse		$\bar{I}_x = \frac{1}{4}\pi ab^3$ $\bar{I}_y = \frac{1}{4}\pi a^3b$ $J_O = \frac{1}{4}\pi ab(a^2 + b^2)$