

PROBLEMA 1

a) tensão no osso.

O osso está à compressão e a tensão é

$$\sigma = \frac{F}{A} \text{ com Área do osso } A = \pi \left( (12.5 \times 10^{-3})^2 - (7.5 \times 10^{-3})^2 \right) = 3.1416 \times 10^{-4}$$

$$\sigma = -\frac{1000}{3.1416 \times 10^{-4}} = -3.183 \times 10^6 \text{ Pa} = -3.183 \text{ MPa}$$

b) A tensão normal em cada uma das placas é

$$\sigma = \frac{F_{\text{PLACA}}}{A_{\text{PLACA}}} \quad F_{\text{PLACA}} = \frac{F}{2}$$

$$\sigma = -\frac{500}{9.5 \times 10^{-5}} = -5.26 \times 10^6 = -5.26 \text{ MPa}$$

c) A tensão de corte nos parafusos.

$$\tau = \frac{F/2}{A_{\text{parafuso}}} = \frac{500}{\pi (2.5 \times 10^{-3})^2} = 25.46 \times 10^6 \text{ Pa} = 25.46 \text{ MPa}$$

d) A deformação axial das placas é

$$\delta = \frac{PL}{EA} = \frac{500 \times (50 \times 10^{-3})}{200 \times 10^9 \times 9.5 \times 10^{-5}} = 1.316 \times 10^{-6} \text{ m} = 1.316 \mu\text{m}$$

PROBLEMA 2

a) A tensão no ponto A é 11.2 MPa de compressão, logo:

$$-11.2 \times 10^6 = -\frac{P}{A} - \frac{My}{I} \quad \text{com } \begin{cases} P = 1000 \text{ N} \\ A = \pi (r_{\text{ext}}^2 - r_{\text{int}}^2) \text{ m}^2 \\ M = 1000 \times 25 \times 10^{-3} \text{ Nm} \\ y = r_{\text{ext}} = 16 \times 10^{-3} \text{ m} \\ I = \frac{\pi}{4} (r_{\text{ext}}^4 - r_{\text{int}}^4) \end{cases}$$

$$-11.2 \times 10^6 (r_{\text{ext}}^4 - r_{\text{int}}^4) = -\frac{1000}{\pi} (r_{\text{ext}}^2 + r_{\text{int}}^2) - \frac{1000 \times 16 \times 10^{-3} \times 25 \times 10^{-3}}{\frac{\pi}{4}}$$

$$-11.2 \times 10^6 r_{\text{ext}}^4 + 11.2 \times 10^6 r_{\text{int}}^4 = -318.31 r_{\text{ext}}^2 - 318.31 r_{\text{int}}^2 - 0.509$$

$$11.2 \times 10^6 r_{\text{int}}^4 + 318.31 r_{\text{int}}^2 - 0.1432 = 0$$

$$11,2 \times 10^6 x^2 + 318,31 x - 0,1432 = 0$$

DUAS RAÍZES SÃO:

$$x = -128,174 \times 10^{-6} \text{ ou } x = 99,7531 \times 10^{-6}$$

A RAÍZ NEGATIVA NÃO PODE SER LOGO

$$r_{int}^2 = 99,7531 \times 10^{-6} \Rightarrow r = 9,987 \times 10^{-3} \text{ mm}$$

$$d_{int} = 19,975 \text{ mm} \approx 20 \text{ mm}$$

A espessura do aço compacto é:

$$e = \frac{32 - 20}{2} = \underline{\underline{6 \text{ mm}}}$$

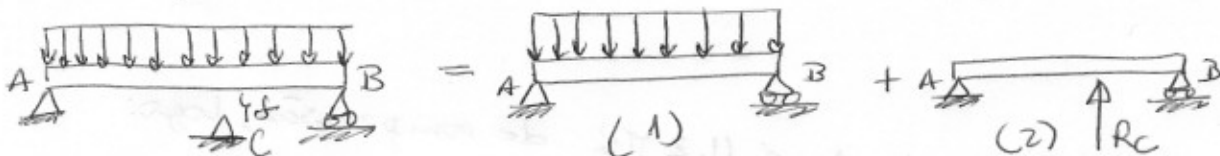
b) Com  $d_{int} = 20 \text{ mm}$

$$\sigma_B = -\frac{1000}{\pi(r_{ext}^2 - r_{int}^2)} + \frac{1000 \times 25 \times 10^{-3} \times 16 \times 10^{-3}}{\frac{\pi}{4}(r_{ext}^4 - r_{int}^4)} = 7,13 \times 10^6 \text{ Pa} = 7,13 \text{ MPa}$$

$$c) \sigma_C = -\frac{1000}{\pi(r_{ext}^2 - r_{int}^2)} = -2,04 \times 10^6 \text{ Pa} = -2,04 \text{ MPa}$$

### PROBLEMA 3

1) O PROBLEMA É ESTATICAMENTE INDETERMINADO. UTILIZANDO O MÉTODO DA SOBREPONÇÃO E COM A AJUDA DAS TABELAS DO FORMALIZADO TEMOS:



$$e = y_{(1)} + y_{(2)} = \delta$$

PROBLEMA (1)

$$y = -\frac{q}{24EI} (x^4 - 2Lx^3 + L^3x)$$

$$y(x=1,4) = -\frac{600}{24EI} (1,4^4 - 2 \times 2 \times 1,4^3 + 2^3 \times 1,4) = -32,26 \times 10^{-3} \text{ m}$$

PROBLEMA (2)

$$y(x=1,4) = \frac{R_C \times 1,4^2 \times 0,6^2}{3 \times 70 \times 10^9 \times 45 \times 10^{-9} \times 2} = 3,73 \times 10^{-5} R_C$$

$$\text{Como } y_{(1)} + y_{(2)} = \delta = -10 \times 10^{-3} \Rightarrow -32,26 \times 10^{-3} + 3,73 \times 10^{-5} R_C = -10 \times 10^{-3} \\ \Rightarrow \underline{\underline{R_C = 596,43 \text{ N}}}$$

OUTRAS REAÇÕES SÃO OBTIDAS DAS EQS. DA ESTATICA,

$$\uparrow \sum F = 0 \quad R_A + R_B + 596,43 - 600 \times 2 = 0$$

$$\uparrow \sum M_A = 0 \quad 596,43 \times 1,4 + R_B \times 2 - 600 \times 2 \times 1 = 0$$

$$\left. \begin{array}{l} R_B = 182,50 \text{ N} \\ R_A = 421,07 \text{ N} \end{array} \right\}$$

$$\underline{\underline{R_A = 421,07 \text{ N}}}$$

b) A DEFORMADA PODE SER DETERMINADA UTILIZANDO AS TABELAS DO FÓRMULARIO:

EXAME 1º ETAPA  
3/07/2006

ENTRE AC

$$y(x) = -\frac{R_c b}{6EI L} [x^3 - (L^2 - b^2)x] - \frac{q}{24EI} (x^4 - 2Lx^3 + L^3x)$$

com  $b = 0,6 \text{ m}$

$L = 2 \text{ m}$

$R_c = 594,93 \text{ N}$

$q = 600 \text{ N/m}$

ENTRE CB

$$y(x) = -\frac{R_c a}{6EI L} [(L-x)^3 - (L^2 - b^2)(L-x)] - \frac{q}{24EI} (x^4 - 2Lx^3 + L^3x)$$

com  $a = 1,4 \text{ m}$

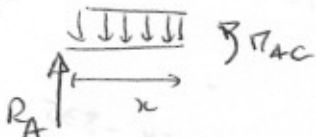
LOGO  $y$  A RGA DA VIGA É:

$$y_{AC}(x=1) = -\frac{596,43 \times 0,6}{6 \times 70 \times 10^9 \times 45 \times 10^{-9} \times 2} [1 - (2^2 - 0,6^2) \times 1] - \frac{600}{24 \times 70 \times 10^9 \times 45 \times 10^{-9}} (1 - 2 \times 2 + 2^3)$$

$$y_{AC}(x=1) = -14,68 \times 10^{-3} \text{ m} = \underline{\underline{-14,68 \text{ mm}}}$$

OU ENTÃO UTILIZANDO INTEGRAÇÃO A PARTIR DA EXPRESSÃO DOS MOMENTOS:

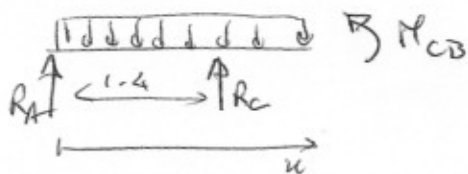
AC



$$M_{AC} - 421,07x + 600 \frac{x^2}{2} = 0$$

$$M_{AC} = -300x^2 + 421,07x$$

CB



$$M_{CB} + 600 \frac{x^2}{2} - 421,07x - 596,43(x - 1,4) = 0$$

$$M_{CB} = -300x^2 + 1017,5x - 835$$

INTEGRANDO 2 VEZES

AC

$$EI \frac{dy_{AC}}{dx} = -300 \frac{x^3}{3} + 421,07 \frac{x^2}{2} + C_1$$

$$EI y_{AC} = -300 \frac{x^4}{12} + 421,07 \frac{x^3}{6} + C_1 x + C_2$$

$$EI \frac{dy_{AC}}{dx} = -100x^3 + 210,53x^2 + C_1$$

$$EI y_{AC} = -25x^4 + 70,18x^3 + C_1 x + C_2$$

CB

$$EI \frac{dy_{CB}}{dx} = -300 \frac{x^3}{3} + 1017,5 \frac{x^2}{2} - 835x + C_3$$

$$EI y_{CB} = -300 \frac{x^4}{12} + 1017,5 \frac{x^3}{6} - 835 \frac{x^2}{2} + C_3 x + C_4$$

$$EI \frac{dy_{CB}}{dx} = -100x^3 + 508,75x^2 - 835x + C_3$$

$$EI y_{CB} = -25x^4 + 169,58x^3 - 417,5x^2 + C_3 x + C_4$$

→

COND. FRONTEIRA

$$x=0 \quad y_{AC} = 0 \Rightarrow c_2 = 0$$

$$x=2 \quad y_{CB} = 0 \Rightarrow -713,36 + 2c_3 + c_4 = 0$$

COMPATIBILIDADE EM  $x=1,4$

$$\frac{dy_{AC}}{dx} = \frac{dy_{CB}}{dx} \Rightarrow 584,48 + c_1 - c_3 = 0$$

$$y_{AC} = y_{CB} \Rightarrow 545,54 + 1,4c_1 - 1,4c_3 - c_4 = 0$$

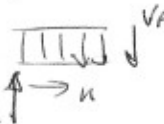
$$\Rightarrow \begin{cases} c_1 = -91,86 \\ c_2 = 0 \\ c_3 = 492,62 \\ c_4 = -272,76 \end{cases}$$

então  $y_{AC} = \frac{1}{EI} (-25x^4 + 70,18x^3 - 91,86x)$

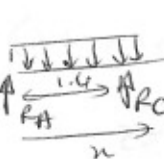
e  $y_{CB} = \frac{1}{EI} (-25x^4 + 169,58x^3 - 417,5x^2 + 492,62x - 272,76)$

e  $y_{AC}(x=1) = -14,68 \times 10^{-3} \text{ m} = -14,68 \text{ mm}$

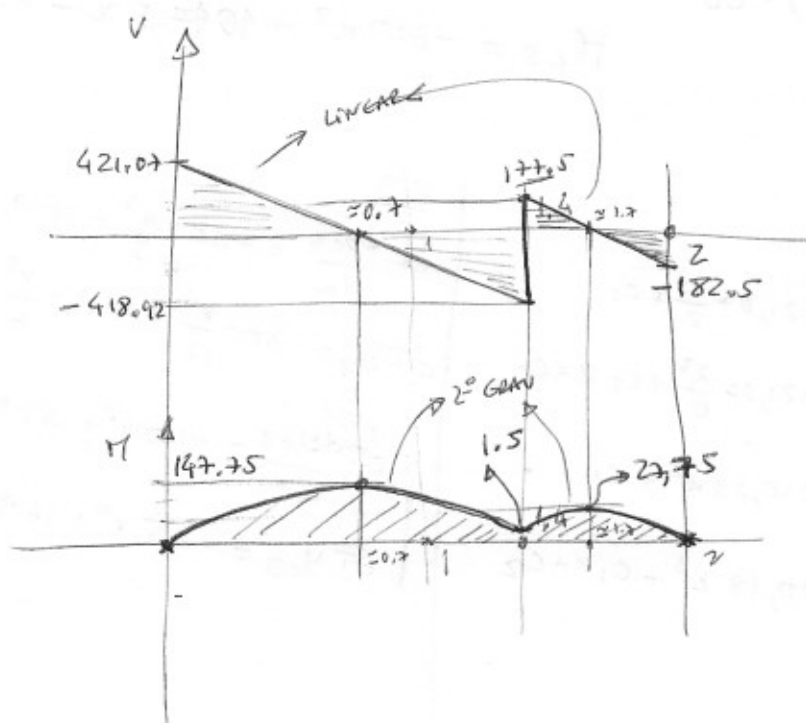
c) Diagramas de  $v$  e  $\pi$

AB   $V_{AB} + 600x - 421,07 = 0$

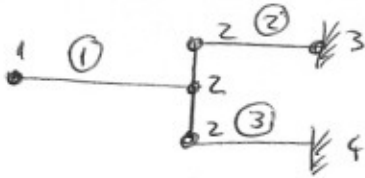
$V_{AB} = -600x + 421,07$        $\pi_{AB} = -300x^2 + 421,07x$

BC   $V_{BC} = -600x + 596,43 + 421,07$

$V_{BC} = -600x + 1017,5$        $\pi_{BC} = -300x^2 + 1017,5x - 835$



PROBLEMA 4



A — B → elemento linear unidimensional (elemento barra)

EXAME 1º/6/2004  
3/07/2006

| ELEMENTO | A <sup>(1)</sup> | B <sup>(2)</sup> |
|----------|------------------|------------------|
| ①        | 1                | 2                |
| ②        | 2                | 3                |
| ③        | 2                | 4                |

a)  $K^{①} = \frac{E^{①} A^{①}}{L^{①}} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$   $E^{①} = 200 \text{ GPa}$   
 $A^{①} = 3.1416 \times 10^{-6} \text{ m}^2$   
 $L^{①} = 0.1 \text{ m}$

$K^{①} = \begin{bmatrix} 62.83 & -62.83 \\ -62.83 & 62.83 \end{bmatrix} \times 10^6$

$K^{②} = K^{③} = \frac{E^{②} A^{②}}{L^{②}} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$   $E^{②} = E^{③} = 200 \text{ GPa}$   
 $A^{②} = A^{③} = 9.5 \times 10^{-5} \text{ m}^2$   
 $L^{②} = 25 \times 10^{-3} \text{ m}$

$K^{②} = K^{③} = \begin{bmatrix} 760 & -760 \\ -760 & 760 \end{bmatrix} \times 10^6$

b) A MATRIZ GLOBAL É 4x4

$K_{11}^G = K_{11}^{①}$   $K_{12}^G = K_{12}^{①}$   $K_{13}^G = 0$   $K_{14}^G = 0$

$K_{22}^G = K_{11}^{②} + K_{11}^{③} + K_{22}^{①}$   $K_{23}^G = K_{12}^{②}$   $K_{24}^G = K_{12}^{③}$

$K_{33}^G = K_{22}^{②}$   $K_{34}^G = 0$

$K_{44}^G = K_{22}^{③}$

$K^{GLOBAL} = \begin{bmatrix} 62.83 & -62.83 & 0 & 0 \\ -62.83 & 2 \times 760 + 62.83 & -760 & -760 \\ 0 & -760 & 760 & 0 \\ 0 & -760 & 0 & 760 \end{bmatrix} \times 10^6$

$K_{22}^G = 2 \times 760 + 62.83 = 1582.83$

c) Não existem forças distribuídas  
forças concentradas só temos no nó 1

$$F^G = \begin{pmatrix} 1000 \\ 0 \\ 0 \\ 0 \end{pmatrix}$$

d)  $u_3 = u_4 = 0$

$$10^6 \times \begin{bmatrix} 62.83 & -62.83 & 0 & 0 \\ -62.83 & 1582.83 & -760 & -760 \\ 0 & -760 & 760 & 0 \\ 0 & -760 & 0 & 760 \end{bmatrix} \begin{pmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \end{pmatrix} = \begin{pmatrix} 1000 \\ 0 \\ 0 \\ 0 \end{pmatrix}$$

Como  $u_3$  e  $u_4 = 0$  posso eliminar as linhas e colunas correspondente e fica

$$10^6 \times \begin{bmatrix} 62.83 & -62.83 \\ -62.83 & 1582.83 \end{bmatrix} \begin{pmatrix} u_1 \\ u_2 \end{pmatrix} = \begin{pmatrix} 1000 \\ 0 \end{pmatrix}$$

e) Resolvendo o sistema

$$\begin{cases} 10^6 \times (62.83 u_1 - 62.83 u_2) = 1000 \\ 10^6 \times (-62.83 u_1 + 1582.83 u_2) = 0 \end{cases}$$

$$\Rightarrow u_1 = 16.57 \times 10^{-6} \text{ m}$$

$$u_2 = 6.57 \times 10^{-7} \text{ m} \quad \rightarrow \delta \text{ de meia peça}$$

para a peça inteira  $\delta = u_2 \times 2 = 1.316 \times 10^{-6} \text{ m}$   
 $= \underline{\underline{1.316 \mu\text{m}}}$